

Worksheet n°5: Feedback control

Let a physical system be governed by the following equations

$$\frac{dw}{dt} = Aw + Bu, y = Cw + g$$

$$A = \begin{bmatrix} 1 & -10 \\ 10 & 1 \end{bmatrix}, B = \begin{pmatrix} -1 \\ -1 \end{pmatrix}, C = (1 \quad 1)$$

where u is a control signal subject to a noise v .

Octave:

```
A=[[1 -10];[10 1]]
B=[-1; -1]
C=[1 1]
eig(A) % eigenvalues of A
sys=ss(A,[B zeros(size(B))],C,[0 1]) % two inputs u and g, one output y
figure(1)
impulse(sys) % plot impulse response
```

A/ Temporal domain

a/ State-feedback and pole placement.

We look for a control law under the form: $u = Kw + v$. The closed-loop system is:

$$\frac{dw}{dt} = Aw + Bu = (A + BK)w + Bv$$

$$y = Cw + g$$

Find K so that the closed-loop eigenvalues (eigenvalues of $A + BK$) are located at $\lambda = -1 \pm i10$.

Octave:

```
K=[9/5 11/5] % gains of controller
eig(A+B*K) % eigenvalues of closed-loop system
sysf=ss(A+B*K,[B zeros(size(B))],C,[0 1]) % two inputs v and g, one output y
figure(2)
impulse(sysf(1,1),'b') % impulse response of closed-loop system from input 1 (v) to output
% 1 (y)
hold on
K2=[10 10] % other control gains
```

```

eig(A+B*K2)          % closed-loop eigenvalues

sysfsf2=ss(A+B*K2,[B zeros(size(B))],C,[0 1]) % two inputs v and g, one output y

impulse(sysfsf2(1,1),'r') % impulse response of closed-loop system from input 1 (v) to output
                        % (1) y

%%

dt=0.01          % sampling time

t=0:dt:1000;    % time samples

S=0.01;         % PSD of white noise

variance=S/dt; % variance of white noise

v=sqrt(variance)*randn(size(t)); % white noise of PSD (S)

[yy,tt]=lsim(sysfsf(1,1),v,t); % generate output y from noise v with close-loop system

figure(3)

plot(tt,v,'g') % plot noise

hold on

plot(tt,yy,'k'); % plot output y of closed-loop system

std(yy) % standard deviation of output

norm(sysfsf(1,1))*sqrt(S) % 2-norm of impulse response times sqrt(PSD)

%%

[yy2,tt2]=lsim(sysfsf2(1,1),v,t); % generate output y from noise v with close-loop system

plot(tt2,yy2,'m'); % plot output y of closed-loop system

std(yy2) % standard deviation of output

norm(sysfsf2(1,1))*sqrt(S) % 2-norm of impulse response times sqrt(PSD)

%%

```

b/ Observer-feedback and pole placement.

Introducing an estimated state for the control input ($u = Kw_e + v$), the governing equation reads:

$$\dot{w} = Aw + BKw_e + Bv, \quad y = Cw + g$$

A dynamic observer may be obtained by determining L such that

$$\dot{w}_e = Aw_e + BKw_e - L(y - y_e), \quad y = Cw + g, \quad y_e = Cw_e.$$

The estimation error $e = w - w_e$ is governed by $\dot{e} = \dot{w} - \dot{w}_e = (A + LC)e + (B - L) \begin{pmatrix} v \\ g \end{pmatrix}$.

L needs to be chosen so that $A + LC$ be stable. Find L so that the eigenvalues of $A + LC$ are located at $\lambda = -1 \pm i10$.

Octave:

```
L=[-11/5; -9/5] % observer gains

eig(A+L*C)      % eigenvalues of equation governing error

sysdo=ss(A+B*K+L*C,-L,eye(size(A)),zeros(2,1)) % one input y, two outputs w1 and w2

figure(4)

impulse(sysdo,'b')          % impulse response of observer

hold on

L2=[-9; -9]          % alternative gains

eig(A+L2*C)          % eigenvalues of equation governing error

sysdo2=ss(A+B*K+L2*C,-L2,eye(size(A)),zeros(2,1)) % one input y, two outputs w1 and w2

impulse(sysdo2,'r')      % impulse response of observer
```

c/ The compensator (combining the estimator and the controller) is given by:

$$\begin{aligned}\dot{w}_e &= (A + BK + LC)w_e - Ly \\ u &= Kw_e + v\end{aligned}$$

Octave:

```
sysc=ss(A+B*K+L*C,-L*zeros(size(L)),K,[0 1]) % compensator, two inputs (y, v), one output
% u

figure(5)

impulse(sysc)          % impulse response

eig(A+B*K+L*C)        % eigenvalues of compensator
```

d/ The closed-loop system is:

$$\begin{pmatrix} \dot{w} \\ \dot{w}_e \end{pmatrix} = \underbrace{\begin{pmatrix} A & BK \\ -LC & A + BK + LC \end{pmatrix}}_{A_{cl}} \begin{pmatrix} w \\ w_e \end{pmatrix} + \underbrace{\begin{pmatrix} B & 0 \\ 0 & -L \end{pmatrix}}_{B_{cl}} \begin{pmatrix} v \\ g \end{pmatrix}, \quad \begin{pmatrix} u \\ y \end{pmatrix} = \underbrace{\begin{pmatrix} 0 & K \\ C & 0 \end{pmatrix}}_{C_{cl}} \begin{pmatrix} w \\ w_e \end{pmatrix} + \begin{pmatrix} v \\ g \end{pmatrix}$$

Octave:

```
Acl=[[A B*K];[-L*C A+B*K+L*C]]

Bcl=[[B zeros(size(B))];[ zeros(size(B)) -L]]

Ccl=[[zeros(size(C)) K];[C zeros(size(C))]]

syscl=ss(Acl,Bcl,Ccl,eye(2,2)) % closed-loop system, two inputs v and g, two outputs u
% and y
```

```

eig(Acl)                % closed-loop eigenvalues
%%
figure(6)
[y,tim]=impulse(syscl);    % impulse responses of closed-loop system
plot(tim(:),y(:,2,1), 'b') % plot impulse response from input 1 (v) to output 2 (y)
hold on
%%
Acl2=[[A B*K2];[-L2*C A+B*K2+L2*C]] % closed-loop system with alternative gains
Bcl2=[[B zeros(size(B))];[ zeros(size(B)) -L2]]
Ccl2=[[zeros(size(C)) K2];[C zeros(size(C))]]
syscl2=ss(Acl2,Bcl2,Ccl2,eye(2,2))
eig(Acl2)
%%
[y2,tim2]=impulse(syscl2,tim) ; % impulse responses of closed-loop system
plot(tim2(:),y2(:,2,1),'r')    % plot impulse response from input 1 (v) to output 2 (y)
%%
[y3,tim3]=impulse(sysssf(1,1),tim); % impulse response of state-feedback controller from
                                     % v to y
plot(tim3(:),y3(:),'k')        % plot impulse response of state-feedback controller from
                                     % input v to output y
%%
norm(syscl(2,1))             % norm of impulse response from input 1 (v) to output 2 (y)
norm(syscl2(2,1))           % same but with alternative compensator
norm(sysssf(1,1))           % same with state-feedback controller
%%
[yy3,tt3]=lsim(syscl(2,1),v,t); % generate output y from noise v with close-loop system
figure(3)
plot(tt3,yy3,'b');          % plot output y of closed-loop system

```

```

hold on;

std(yy3)                % standard deviation of output

norm(syscl(2,1))*sqrt(S) % 2-norm of impulse response times sqrt(PSD)

%%

[yy4,tt4]=lsim(syscl2(2,1),v,t); % same with alternative compensator

plot(tt4,yy4,'r');      % same with alternative compensator

std(yy4)                % standard deviation of output

norm(syscl2(2,1))*sqrt(S) % 2-norm of impulse response times sqrt(PSD)

%%

pkg load all;

[spectra,freq]=pwelch(yy3,1000,0.5,1000,1/dt);% compute frequency spectrum of output y
                % MATLAB: [spectra,freq]=pwelch(yy3,1000,500,1000,1/dt);

figure(7)

loglog(freq*2*pi,abs(spectra)) % plot frequency spectrum

```

B/ Frequency space

a/ Physical system.

Take the Laplace transform of the equations governing the physical system:

$$\frac{dw}{dt} = Aw + Bu, \quad y = Cw + g$$

and determine the transfer functions between (u, g) and y .

The transfer-function between u and y is denoted $P(s)$. What are the poles and zeros of this transfer function?

Compare the result with Octave:

```

tf(sys)

P=tf(sys(1,1)) % open-loop transfer function P(s) between u and y.

pole(P)        % poles of P(s)

zero(P)        % zeros of P(s)

figure(8)

bode(P)        % Plot magnitude and phase of transfer function between u and y.

```

b/ What is the transfer-function $K(s)$ of the compensator (defined in A/c/) between y and u ?

Octave:

```
tf(sysc)
K=tf(sysc(1,1)) % transfer-function  $K(s)$  of compensator between  $y$  and  $u$ .
pole(K)      % poles of  $K(s)$ 
zero(K)      % zeros of  $K(s)$ 
figure(9)
bode(K)      % Plot magnitude and phase of transfer function between  $y$  and  $u$ .
```

c/ Closed-loop system

```
cltf=tf(syscl) % 4 transfer functions of closed-loop system
get(cltf)      % properties of cltf
cltf(1,1).num  % numerator of closed-loop transfer function from  $v$  to  $u$ 
cltf(1,1).den  % denominator of closed-loop transfer function from  $v$  to  $u$ 
pole(cltf(1,1)) % poles of closed-loop transfer function from  $v$  to  $u$ 
pole(cltf(1,2)) % poles of closed-loop transfer function from  $g$  to  $u$ 
pole(cltf(2,1)) % poles of closed-loop transfer function from  $v$  to  $y$ 
pole(cltf(2,2)) % poles of closed-loop transfer function from  $g$  to  $y$ 
zero(1-P*K)    % should be compared to previous poles
figure(10)
bode(cltf(1,1)) % bode plot from from  $v$  to  $u$  % control cost assessment
figure(11)
bode(cltf(2,1)) % bode plot from from  $v$  to  $y$ 
                % performance assessment, compare to figure 7
```

d/ Robustness

Find largest a satisfying $0 < a < 1$, such that $1 - aPK$ exhibits a zero with a positive real part.

Find smallest a satisfying $a > 1$, such that $1 - aPK$ exhibits a zero with a positive real part.

Find smallest ϕ satisfying $\phi > 0$, such that $1 - e^{i\phi}PK$ exhibits a zero with a positive real part.